AMENDMENTS TO THE SPECIFICATION:

Please replace the paragraph on page 1, beginning on line 15, with the following amended paragraph.

Indirectly heated rotary kilns typically comprise a rotating tube, generally surrounded by a bowl-shaped heating tunnel. Usually, the rotating ~~rube~~ tube is not completely surrounded by the tunnel, but rather projects from the tunnel on both front sides. Within the heating tunnel, heated gas is conducted through the heating tunnel, in order to provide the energy for the chemical and/or thermal processes taking place in the rotating tube. To this end, the heating tunnel has one or more inlets and outlets for the heating medium.

Please replace the paragraph on page 2, beginning on line 22 with the following amended paragraph.

German Patent No. 30 47 404 A1 describes a ring seal for a rotary kiln. The ring seal is provided as a stationary ring, which is made from a number of graphite parts, which, in part are arranged in the form of a ring, disposed alongside one another. The graphite parts are held with their inner side in place against a cylindrical rotating ~~rube~~ tube surface, which is coaxial with respect to the rotary kiln. The graphite parts rotate with the rotating tube. The pressing is achieved by a cable loop under tension at its ends, which surrounds the graphite units. The rotary kiln described in this publication, however, is not an indirectly heated rotary kiln, so that the ring seal is exposed to far lower temperatures and temperature differences. Furthermore, here the application pressure must also correspond to the aforementioned application pressure because of the material, i.e. graphite.

Please replace the paragraph on page 6, beginning on line 22 with the following amended paragraph.

A significant feature of the light construction sealing materials described herein is that such materials do not exert excessive abrasion forces on the rotating tube. In a preferred embodiment of the ring sealing assembly, the sealing segments are made of

material which has a polishing effect on the rotating tube. In this way, the rotating tube is less impaired, as compared to currently known sealing assemblies. Furthermore, use of the present invention ring sealing assemblies reduce the surface roughness of the sealing surface of the rotating ~~rube~~ tube, which improves the sealing and thus further reinforces the seal tightness. As a result of the reduced surface roughness of the rotating tube, the wear of the sealing assembly is also reduced, which further increases or improves the service life and sealing characteristics of the sealing assembly. A carbon fiber felt is particularly preferred as a material for this purpose.

Please replace the paragraph on page 9, beginning on line 7 with the following amended paragraph.

Although not necessarily to scale, Figure 6 is a longitudinal sectional view through the preferred embodiment of an indirectly heatable rotary tubular kiln with a preferred embodiment sealing assembly installed therein. A first holder 42 is placed on a heating tunnel 40, on its front end side. This holder is separated from a second holder 44 by a spacer 46. The two holding rings are thereby positioned to a certain radial distance from the rotating tube 20. The spacer 46 is positioned in such a way, at a distance from the rotating tube, that there is sufficient space for the sealing segments 10 and the application pressure element 30 between the rotating tube and the spacer 46. If the spacer, as shown in Figure 6, is somewhat wider than the sealing segments 10, the sealing segments 10 are pressed against the heating tunnel-side holding rings 42 by means of one or more application pressure elements. This can be done, for example, via one or more adjustment screws 48 arranged in the holding ring 44. However, all other application pressure systems which would enable a sealing placement of the sealing on a front surface are also included within the present invention. Thus, for example, the width of the spacer 46 can be dimensioned exactly so that it corresponds to the width of the sealing segments. If the sealing segment material can be compressed, the width of the sealing in operation can also be determined by the width of the spacer 46, in that the sealing segments are pressed in between the holding rings. In addition, the holder 42 can be part of the heating tunnel 40. The sealing segments 10 must, in any case, be positioned on the heating tunnel 40, so that the

heating medium found in its interior, which flows toward the sealing segments 10 along the rotating tube axis in the direction of arrow B, cannot flow laterally around the sealing segments and thus cannot be introduced into the environment 52. Figure 6 also illustrates in a detailed magnified portion, an optional cover 16 disposed on the sealing ring comprised of sealing segments 10. The cover 16 defines a plurality of apertures 18 adapted to facilitate the removal of debris from the sealing assembly.